2.1 NAME OF COMPONENTS



2.4 TRANSPORTATION DIMENSION AND WEIGHT

OVERALL DIMENSIONS OF MACHINE ON A TRAILER

OVERALL DIMENSIONS OF A COMPLETE MACHINE ON A TRAILER

ltem	Туре	2.07 M (6 ft-9 in) Arm + 600 mm (23.6 in) shoes	2.07 M (6 ft-9 in) Arm + 450 mm (17.7 in) shoes
Width	m (ft-in)	2,470 (8'1")	2,320 (7'7")
Weight	kg (lb)	6,920 (15,260)	6,700 (14,770)

Specification :

- The parenthesis shows 2.07m arm model.
- Regarding 2.07m arm machine, put the arm cylinder rod in a transport position (B) by referring to the outside dimensions in 2.2 MACHINE DIMENSIONS.



OVERALL DIMENSIONS OF WITHOUT ARM AND BUCKET

Туре	A : Shoe width	L : Length	H : Height	W : Width	Weight
Combination	mm (ft-in)	mm (ft-in)	mm (ft-in)	mm (ft-in)	mm (ft-in)
Without arm and bucket	600 (23.6")	5,670 (18'7")	2,600 (8'6")	2,470 (8'1")	6,440 (14,200)
Without and and bucket	450 (17.7")	5,670 (18'7")	2,600 (8'6")	2,320 (7'7")	6,220 (13,715)



11.10 COUNTERWEIGHT LIFTING JIG

Bracket for left side





Material : Mild steel

Fig. 11-3 Counterweight lifting bracket



21.4 ARM IN / OUT CONFLUX MECHATRO CONTROL SYSTEM

- (1) Start arm in and out operation simultaneously, and operation pilot pressure switches arm spool, and is input into low pressure sensor.
- (2) The voltage output by low pressure sensor is input into mechatro controller, and the controller processe the pilot signal and outputs the command corresponding to input voltage to P1 by-pass cut valve.
- (3) P1 by-pass cut valve outputs pilot secondary pressure corresponding to the command output by mechatro controller, and switches P1 by-pass cut valve of control valve.
- (4) By switching arm main spool with operation spool pressure and switching P1 by-pass cut valve according to mechatro control command, the oil delivered by P1 and P2 pumps is confluxed during arm operation.

24.1 HYDRAULIC COMPONENTS

24.1.1 HYDRAULIC PUMP & REGULATOR

24.1.1.1 SUMMARY

(1) General view and hydraulic port





(2) Specifications

Table 2	4-1
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Specifications			Spec. 1	Spec. 2		
Parts No.			YT10V00008F1	YT10V00009F1		
Model			K3SP36B-101R-1002	K3SP36B-101R-2002		
Main	Displacement capacity cm ³ /rev		31.4 X 2	←		
pump	Max. pressure	MPa (psi)	29.4 (4270)	←		
	Max. flow	_/min (gal/min)	66 (18) X 2	←		
	Purpose of use	•	Pilot hydraulic pressure source (A3)	Dozer pressure source (A5)	Pilot hydraulic pressure source (A3)	
Attached	Displacement capacity cm ³ /rev		9.5	23.1	9.5	
gear pump	Max. pressure MPa (psi)		3.4 (500)	19.6 (2840)	3.4 (500)	
	Max. flow L/min (gal/min)		20 (5.3)	48.5 (13)	20 (5.3)	
Revolutio	า	(min ⁻¹) rpm	Rate : 2100 (Clockwise as seen from the shaft), high idle : 2310			
Weight kg (lbs)		41 (90)	44 (88)			
Oil quantity L (gal)		1.0 (0.26)	←			
Control characteristics of regulator			Simultaneous total horsepower, Power shift control of air-conditioner, Variable horsepower control	Simultaneous total horsepower, Power shift control of air-conditioner, Power shift control of the No.3 pump Variable horsepower control		

(11)Function of over road relief valve

 Pressurized oil P is filled up in chamber A through the inside of the piston built in the plunger G and choke B. The plunger G and the socket are sealed securely. The socket is sealed securely against the poppet as well.



2) When the hydraulic pressure at port P arrives at a set value of spring C, it pushes the piston which in turn opens the poppet F.

On that occasion, the hydraulic pressure goes through the inside of the piston \rightarrow choke B \rightarrow chamber A \rightarrow annular orifice D and drilled hole E goes around the outer circumference of the socket and flows out to the tank passage.



3) If the poppet F opens, the pressure of chamber A falls. Then the plunger G opens and the pressure of port P flows directly to the tank T passage through drilled hole H.





4) If the pressure at port P falls below a set value of spring C, the poppet F is pressed against the seat by the action of spring C. As the result, the pressure of chamber A equals that of port P. This presses the plunger G against the socket seat and brings it back where it was. (Fig. 24-54)



Fig. 24-57

5) Function of anti-cavitation

This function supplies oil from the T (tank) path in case negative pressure occurs at port P. When the tank path pressure gets higher than that of port P, the socket is subject to a force that pushes it up. This creates a space between the body seat and the socket. The oil from the T path (tank) flows into port P, and fills up the space.

24. COMPONENTS SYSTEM



- (5) Pressure Switch (Attached to Receiver Dryer)
 - 1) Contents of checking (specification)

Low pressure side OFF pressure : 0.2 ± 0.02 MPa (28 ± 3 psi)

Low pressure side return pressure : OFF pressure + 0.03 MPa (4.3 psi) or below

High pressure side OFF pressure : 3.1 ± 0.2 MPa (455 ± 28 psi)

High pressure side return pressure : OFF pressure -0.59 \pm 0.02 MPa (-85 \pm 2.8psi)



Fig. 25-38 Pressure switch

LOW PRESSURE F	RETURN HI	GH PRESSURE RETUR	RN MPa (psi)
(26) (30) (31) (3	ר 36) (31	lo) (4	1 30) (480)
0.18 0.21 0.22 0	.25 2.1	16 2.	94 3.33
	<u> </u>		
LOW PRESSURE OFF		HIGH	PRESSURE OFF

2) Performance

Low pressure side :

To prevent actuation of the compressor clutch where refrigerant is deficient

High pressure side :

To prevent actuation of the compressor clutch where pressure rises abnormally because of the clogging of the refrigeration circuit, etc.

- (6) Thermistor (C19)
 - 1) Contents of checking (specification) Resistance across terminals

at 0°C (32°F) : 7.2 K1/2 at 25°C (77°F) : 2.2 K1/2

Note

Normally check that there is no shortcircuit or disconnection.

2) Performance

In order to prevent freezing of the evaporator, the blowoff air temperature of the evaporator is controlled by turning the compressor clutch ON and OFF. The thermistor operates as the sensor of the control.





32.2.1.4.2 ASSEMBLING ROD COVER ASSY

- (1) Press fit bushing (4) in rod cover (3) applying plate on the section. After press-fitting, check that the bushing is not projected from the end surface (A). (See Fig. 32-46)
- (2) Fit snap ring (5).
- (3) Fit backup ring (8) on U-ring groove.
- (4) Apply hydraulic oil on U-ring (7) and fit it on the U-ring groove.

Note

- The U-ring is harder than other seals, so fit it in the groove by hand first, then press in with pushing bar until it is fitted with a click.
- Fitting U-ring paying attention to the direction.
- After fitting backup ring (8) and U-ring (7), check that they are free from the permanent set.



Fig. 32-46 Pressing in bushing



Fig. 32-47 Fitting U-ring



Fig. 32-48 Installing buffer ring assy (6)

(5) Installing buffer ring assy Buffer ring assy (6) is equipped

Buffer ring assy (6) is equipped with square ring and slide ring.

- 1) Fit square ring on the groove.
- 2) Depress slide ring in U shape by hand to house it in groove, and fit it pushing out to periphery.

Note

- Depress it R 6 mm (0.24 in) or more, because if the depressed R is too small, it may remain wrinkled.
- After fitting, check that it is free from wrinkles.
- Fit it giving attention to the seal fitting direction. Reverse fitting produces high pressure between U-ring and seal and may cause the deformation of rod cover.
- Fit slide ring so that the groove matches to the root section of square ring.

(4) Remove guard (B4)

- 1) Remove three capscrews (B19) M8.
- 2) Remove guard (B4).

••••: 13 mm

- (5) Remove water hose (B6)
 - 1) Loosen the hose band (B12).
 - 2) Remove hose (B6).

(6) Remove main radiator (C1) Remove four capscrews (A3) M12 \times 30.



- (7) Removing connection coupling for hydraulic oil (Oil cooler)
 - 1) Remove 4 bolts M8 of coupling (17) just below the radiator.
 - 2) Put oil pan under the coupling (17), remove coupling and let oil out.
- (8) Disconnecting hydraulic oil pipe
 - 1) Shift radiator outward.
 - 2) Plug up pipe ends with rag, etc.
- (9) Disconnecting water lower hose (B5)
 - 1) Loosen the hose band (B13).
 - 2) Pulling out hose (B5).

Note

Remove the hose while sliding the radiator little by little.

(10)Removing radiatorRemove radiator assy by power of men or with crane.Weight : 48 kg (106 lbs)



Fig. 33-64 Removing radiator



Fig. 33-65 Removing coupling





Standard	Remarks
Deplace with a complete set of pilot value when	
Replace with a complete set of pilot valve when	
the amount of leakage reaches more than 1000	Primary pressure : 2.94MPa (427 psi)
cc/min (61 cu•in/min) or 2000 cc/min (122 cu•in/	Oil viscosity : 23 mm²/s
min) at the neutral position of the handle or dur-	
ing operation, respectively.	
Replace with a complete set of pilot valve when	The wear condition to the left is con-
an amount of wear at the sliding section is more	sidered to correspond to the above
than 10 μ m (0.0004") in comparison with the	amount of leakage.
non-sliding section.	
Replace when a wear amount of the tip is more	than 1 mm (0.04").
Replace when a play more than 2 mm(0.079")	A play generated by loosening of tight-
due to wear and so on is found on the disk (302)	ening portion should be adjusted.
or joint section (301) of the operation section.	
Replace with a complete set of pilot valve when	
abnormal noise, hunting or primary pressure	
drop is generated during operation and the trou-	
ble cannot be remedied according to Section	
33.2.3.6 TROUBLESHOOTING.	
	StandardReplace with a complete set of pilot valve when the amount of leakage reaches more than 1000 cc/min (61 cu•in/min) or 2000 cc/min (122 cu•in/ min) at the neutral position of the handle or dur- ing operation, respectively.Replace with a complete set of pilot valve when an amount of wear at the sliding section is more than 10 μ m (0.0004") in comparison with the non-sliding section.Replace when a wear amount of the tip is more than 10 μ m (0.0004") in comparison with the non-sliding section.Replace when a wear amount of the tip is more1 mm $(0.04")$ Network and so on is found on the disk (302) or joint section (301) of the operation section.Replace with a complete set of pilot valve when abnormal noise, hunting or primary pressure drop is generated during operation and the trou- ble cannot be remedied according to Section 33.2.3.6 TROUBLESHOOTING.

33.2.3.5 Maintenance standard

Note

Replace seal such as O-ring with new ones after every disassembly.

33.2.3.6 TROUBLESHOOTING

Phenomenon	Possible Cause	Corrective action		
Secondary pressure	1) Primary pressure is insufficient.	1) Secure primary pressure.		
does not rise.	2) Springs (241-1, 241-2) are broken or fatigued.	2) Replace with new ones.		
	 Clearance between spool (201-1, 201- 2) and valve body (101) is abnormally large. 	3) Replace assembly.		
	4) Play of handle portion is too much.	 Disassembly and reassembly or re- place handle portion. 		
Secondary pressure is	1) Sliding parts are caught.	1) Correct.		
unstable.	2) Tank line pressure varies.	2) Return directly to oil tank.		
	3) Air has contained into pipeline.	3) Release air.		
Secondary pressure is	1) Tank line pressure is high.	1) Return directly to oil tank.		
high.	2) Sliding parts are caught.	2) Correct.		

34.2 DISASSEMBLING AND ASSEMBLING

34.2.1 TRAVEL MOTOR

34.2.1.1 CONSTRUCTION

(1) General view



VIEW Z



HYDRAULIC CIRCUIT



PORT NAME AND SIZE

No.	Ports	Size	
А	Hydraulic oil filling / drain port	DE 1/2	
В	Hydraulic oil filling / drain port	FF 1/2	
D1	Drain port		
D2	Drain port	PF 3/0	
Р	1st / 2nd speed select port	PF 1/4	

ROTATIONAL DIRECTION (Viewed from Z side) INLET B PORT, OUTLET A PORT, CW INLET A PORT, OUTLET B PORT, CCW

Fig. 34-92

41.2 TABLE OF ACTUATOR FOR TROUBLESHOOTING : MECHATRO CONTROL

				E	lectr	ic				Hyd	raulic
	Actuator	Low pressure sensor	P2 bypass cut proportional valve	Travel straight proportional valve	P1 bypass cut proportional valve	Swing P / B solenoid valve	Travel 1-2speed solenoid valve	Safety lock lever solenoid valve	Travel boost-up solenoid valve		
	Opeating condition	S E 1 ~ 10	P S V-B	P S V-C	P S V D	S V 1	S V 3	S V 4	S V 8	P1 pump	P2 Pump
	Boom up inching operation	\bigcirc						\bigcirc		\bigcirc	
	Boom up full lever operation	0	0					0		0	0
	Boom down operation	0						0		0	
	Arm out inching operation	0						0			0
eratio	Arm out full lever operation	0			0			0		0	0
nt op	Arm in inching operation	0				0		0			0
ende	Arm in full lever operation	0			0	0		0		0	0
denp	Bucket digging	0						0		0	
<u> </u>	Bucket dump	0						0		0	
	Swing operation	0	0			0		0			0
	Travel LH / RH operation (1st. speed)	0						0	0	0	0
	Travel LH / RH operation (2nd. speed)	\bigcirc					0	0	0	\bigcirc	0
	Travel right operation + Boom operation	0	\bigcirc	0				\bigcirc	0	\bigcirc	0
	Travel right operation + Bucket operation	0	0					0	0	\bigcirc	0
u	Travel right operation + Arm operation	0			0			0	0	0	0
eratic	Travel right operation + Swing operation	0	\bigcirc			\bigcirc		\bigcirc	0	\bigcirc	0
led of	Travel left operation + Boom operation	0	0					0	0	0	0
mbin	Travel left operation + Bucket operation	0						0	0	0	0
Ŭ	Travel left operation + Arm operation	0		0	0	0		0	0	0	0
	Travel left operation + Swing operation	0	0	0	0	0		0	0	0	0
	Travel LH / RH operation + Boom up operation	0	0	0	0			0	0	\bigcirc	0

42. TROUBLESHOOTING (HYDRAULIC SYSTEM)

(3)-2				
Bucket close power is poor.				
No-load independent operating speed is within a standard value.	n NO			Refer to "Only bucket motion is slow."
YES				
Relief pressure is approx. 29.4MPa (4270psi) when bucket close lever is shifted to the full.	NO	Adjust overload relief valve on bucket close side.	NG	Replace overload relief valve at port A3 (bucket head side).
YES				
close) is approx. 3.43MPa (500psi).				Inspect or replace pilot valve.
YES				Inspect or replace bucket
Bucket spool return spring is broken.				spool assy and pilot cover.
NO	_		_	
Bucket spool moves smoothly by hand. (Remove pilot cover.)	NO	Outer circumference of bucket spool is scored.	YES	Replace bucket spool assy.
YES		NO		NG
				Replace control valve.
				Replace control valve or pump.
(3)-3 Boom up operating power is poor.				
No-load independent operating speed is within a standard value.	NO			Refer to "Only boom raise motion is slow."
YES				
Relief pressure of both P1 and P2 pumps is approx. 29.4MPa (4270psi) when boom raise lever is shifted to the full.	NO			Inspect or replace overload relief valve at port A2 (boom head side).
YES				
Secondary pilot pressure at port Pa2 (boom raise) is approx. 3.43MPa (500psi).	NO			Inspect or replace pilot valve.
YES				
Boom spool return spring is broken.	YES			Inspect or replace boom spool assy and pilot cover.
NO	- 10			
Boom spool moves smoothly by hand.	NO	Outer circumference of boom spool is scored.	YES	Replace boom spool assy.
YES		NO		NG
				Replace control valve.
				Inspect or replace control valve or pump.

Fuel Injection System

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N·m (kgf·m/lb.ft)

